

# Formula Reference Sheet

## Convert pressure to head

$$h = \frac{p}{\rho \cdot g}$$

Where:

h = head, feet or meters

p = pressure, psi (Pa)

$\rho$  = liquid density, lbm/ft<sup>3</sup> (kg/m<sup>3</sup>)

g = Acceleration due to gravity, ft/s<sup>2</sup> (m/s<sup>2</sup>)

## Convert pressure (psi) to head in feet

$$h(\text{ft}) = \frac{p(\text{psi}) * 2.31}{\text{Specific Gravity}}$$

1 meter (m) = 3.281 feet (ft)

## Calculation of velocity head

$$h_v = \frac{v^2}{2g}$$

Where:

$h_v$  = velocity head, ft (m)

v = flow velocity, ft/s (m/s)

g = acceleration constant due to gravity

## Gravitational acceleration constants

$$g = 32.2 \frac{\text{ft}}{\text{s}^2} = 9.81 \frac{\text{m}}{\text{s}^2}$$

## Pump affinity rules with respect to impeller diameter

$$Q_2 = Q_1 \frac{D_2}{D_1} \quad H_2 = H_1 \frac{D_2^2}{D_1^2} \quad P_2 = P_1 \frac{D_2^3}{D_1^3}$$

Where:

Q = flow rate

H = total head

P = pump input power

D = impeller diameter

## Pump affinity rules with respect to pump speed

$$Q_2 = Q_1 \frac{n_2}{n_1} \quad H_2 = H_1 \frac{n_2^2}{n_1^2} \quad P_2 = P_1 \frac{n_2^3}{n_1^3}$$

Where

Q = flow rate

H = total head

P = pump input power

n = pump rotational speed

## Calculation of Net Positive Suction Head Available (NPSHA)

$$NPSHA = h_{atm} + h_{gs} + h_{vs} + z_s - h_{vp}$$

Where

$h_{atm}$  = atmospheric pressure head, ft (m)

$h_{gs}$  = suction gauge head, ft (m)

$h_{vs}$  = suction velocity head, ft (m)

$z_s$  = elevation from the suction gauge centerline to datum, ft (m)

$h_{vp}$  = liquid vapor pressure head, ft (m)

## Calculation of Net Positive Inlet Pressure Available (NPIPA)

$$NPIPA = p_{s,absolute} - p_{vp,abs}$$

Where

$p_{s,absolute}$  = Total suction pressure absolute

$p_{vp,abs}$  = Liquid vapour pressure absolute

## Calculation of electrical input power

$$\text{Input power (kW)} = \frac{I \times V \times PF \times C}{1000}$$

Where

I = current in amperes (A) (meter reading)

V = volts (meter reading)

PF = power factor (motor curve or measured)

C = 1 for single-phase current

= 2 for two-phase four-wire control

= 1.73 for three-phase current

## Calculation of power

### Hydraulic Power as a function of Head

hydraulic power ( $P_u$ ) =  $\frac{Q \times H \times s}{3960}$  (hp), also known as Water Horsepower (WHP,  $P_w$ ) when water is the pumped fluid

Hydraulic power ( $P_u$ )      hp (horsepower)

Head (H)                      feet

Flow (Q)                      gpm

Specific gravity (s)        dimensionless

3960 is a constant that incorporates two conversions: (1) convert flow (gpm) to pounds/minute, and (2) convert the product of mass flow  $\times$  head (pound-foot/minute) to horsepower

$$P_u = \frac{Q \times H \times \rho \times g}{1000} (kW)$$

Where:

$P_u$  = Hydraulic power (kW)

Q = Flow rate (m<sup>3</sup>/s)

H = Head (m)

$\rho$  = Density of fluid (kg/m<sup>3</sup>)

g = Acceleration due to gravity (9.81 m/s<sup>2</sup>)

$$\text{Hydraulic Power } (P_u) = \frac{Q \times H \times s}{0.1022} (kW)$$

Where:

$P_u$  = Hydraulic power (kW)

Head (H) meters (m)

Flow (Q) m<sup>3</sup>/s

Specific gravity (s) dimensionless

0.1022 is metric units conversion to provide power in kilowatt (kW)

### Hydraulic power as a function of total differential pressure

$$P_u (kW) = \frac{Q \left( \frac{m^3}{h} \right) \times \Delta p (kPa)}{3600 \left( \frac{s}{h} \right)} = Q \left( \frac{m^3}{s} \right) \times \Delta p (kPa)$$

$$P_u (hp) = \frac{Q (gpm) \times \Delta p (psi)}{1714}$$

Pump Power Input ( $P$ ) =  $\frac{P_u}{\eta}$ , also known as shaft power or Brake Horsepower (BHP)

Where:

$P_u$  = Hydraulic power (kW)

Pump Efficiency ( $\eta$ )

Power (kW) = 0.746 × Horsepower

## Area of pipe

$$A = \pi \times r^2$$

Where:

A = crosssection area of the pipe inside diameter, ft<sup>2</sup> (m<sup>2</sup>)

r = radius of the pipe inside diameter, ft (m)

Velocity (v) in pipe

$$v = \frac{\text{Volume rate of flow}}{\text{Pipe inside diameter area}}$$

$$v \left( \frac{ft}{s} \right) = \frac{0.4085 \times gpm}{\text{pipe ID}^2 \text{ (inches)}}$$

## Resistance to flow in pipes and fittings

$$h_f = f \times \frac{l}{d} \times \frac{v^2}{2g}$$

Where

$h_f$  = frictional resistance in head

$f$  = piping friction factor

$l$  = length of pipe, ft (m)

$d$  = average ID of piping, ft (m)

$v$  = average velocity, ft/s (m/s)

$g$  = acceleration due to gravity

$$h_f = k \times \frac{v^2}{2g}$$

Where

$h_f$  = frictional resistance in head

$k$  = resistance coefficient for valve or fitting

$v$  = average velocity, ft/s (m/s)

$g$  = acceleration due to gravity